

CFB SIM: A QUASI-1D MODEL OF BIOMASS CO-FIRING IN CIRCULATING FLUIDIZED BED

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Product sheet

Outline

Mathematical Model

Numerical Solution and Results

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Numerical Solution and Results

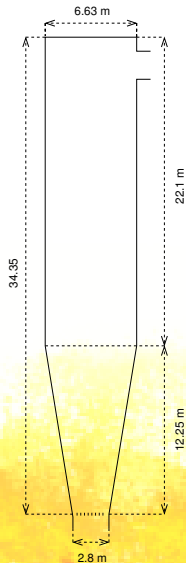
Model properties

Simulated phenomena

- **quasi-1D nozzle multi-phase flow**
- primary, secondary & tertiary **air inflow**
- fuel and bed material **injection**
- fuel and bed material **recirculation**
- fuel **devolatilization** and **burnout**
- particle **attrition**
- **heat** production and transfer

Expected results

- **local**: vertical solids distribution, temperature
- **global**: bed inventory, heat production
- **general**: response to control



Mass balance

$$\frac{\partial(\rho_i \varepsilon_i)}{\partial t} + \frac{1}{A} \frac{\partial(A \rho_i \varepsilon_i U_i)}{\partial x} = \mathcal{M}_i(t, x) + \mu_i,$$
$$\frac{\partial(\rho_g \varepsilon_g Y_X)}{\partial t} + \frac{1}{A} \frac{\partial(A \rho_g \varepsilon_g U_g Y_X)}{\partial x} = \mathcal{M}_X(t, x) + \mu_X$$

for $i \in \{\text{gas, solid (bed material), coal, biomass}\}$ and $X \in \{\text{O}_2, \text{VM}_c, \text{VM}_b\}$

- $\varepsilon_i, \rho_i, U_i$ volume fraction, velocity, and density of phase i
- Y_X mass fraction of species X in the flue gas
- $\mathcal{M}_g, \mathcal{M}_i$... injection of the secondary air, bed material, and fuels
- $\mathcal{M}_{\text{VM}_c}, \mathcal{M}_{\text{VM}_b}$ inflow of VM as part of the fuel inflow
- $-\mu_c, -\mu_b$ mass loss rate due to char particle burnout
- μ_g ... production rate of flue gas due to combustion of char
- $-\mu_{\text{O}_2}$ rate of oxygen consumption by combustion
- $\mu_{\text{VM}_c}, \mu_{\text{VM}_b}$ burnout rate of coal and biomass VM

Passive particle transport

$$\frac{\partial n_i}{\partial t} + \frac{1}{A} \frac{\partial (An_i u_i)}{\partial x} = \mathcal{M}_{n_i}(t, x) + \mu_{n_i}$$

n_i number of particles per unit volume

\mathcal{M}_{n_i} rate of particle number change due to inflow

μ_{n_i} rate of particle number change due to attrition

- for spherical particles, particle diameter is $d_{p,i} = \sqrt[3]{6\varepsilon_i / (\pi n_i)}$.

Momentum balance

$$\frac{\partial(\rho_i \varepsilon_i u_i)}{\partial t} + \frac{1}{A} \frac{\partial(A \rho_i \varepsilon_i u_i^2)}{\partial x} = -\frac{\partial P_i}{\partial x} + \sum_{k \in \{g, s, c, b\}} \beta_{ki} (u_k - u_i) - \frac{2f_i \varepsilon_i \rho_i u_i^2}{D} + R_i \varepsilon_i g + \mathcal{P}_i(t, x) + \pi_i$$

for $i \in \{g, s, c, b\}$

- P_g pressure of gas and for $i \in \{s, c, b\}$, we put $\frac{\partial P_i}{\partial x} = G \frac{\partial \varepsilon_i}{\partial x}$
- $G(\varepsilon_g)$ *solids stress modulus*
- f_i, β_{ki} wall friction and inter-phase friction forces
- $R_i \varepsilon_i g$ gravity / buoyancy term with $R_g = \rho_g$
and $R_i = \rho_i - \rho_g$ for $i \in \{s, c, b\}$
- π_i inter-phase momentum transfer
- \mathcal{P}_i momentum source term

Energy balance

- local inter-phase thermal equilibrium assumed
⇒ single (summed) equation for internal energy in terms of temperature T

$$\sum_{i \in \{g,s,c,b\}} \rho_i \varepsilon_i c_{p,i} \left(\frac{\partial T}{\partial t} + u_i \frac{\partial T}{\partial x} \right) = \frac{1}{A} \sum_{i \in \{g,s,c,b\}} \left(-P_i \frac{\partial (A u_i)}{\partial x} + \varepsilon_i \frac{\partial}{\partial x} \left(A \lambda_i \frac{\partial T}{\partial x} \right) \right) + \sum_{i \in \{g,s,c,b\}} \left(\mathcal{E}_i - u_i (\mathcal{P}_i + \pi_i) + (\mathcal{M}_i + \mu_i) \left(\frac{u_i^2}{2} - \int_0^T c_{p,i}(\tau) d\tau \right) \right) + \dot{Q}$$

where \dot{Q}, \mathcal{E}_i are the total heat and internal energy source terms, respectively

Momentum transfer

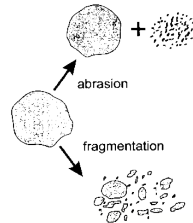
- friction between the gas phase and the solid phases β_{gi} , $i \in \{s, c, b\}$ (*Gidaspow*)
- friction between two solid phases β_{ik} , $i, k \in \{s, c, b\}$ (*Syamlal*)
- wall friction f_i (*Gidaspow*)

Attrition 1

Bed material (long term) attrition (*Tomeczek, Mocek*)

Particle mass decrease rate

$$\frac{dm_s}{dt} = -k_a f(\phi_s) \frac{H_e}{d_h} (u_g - u_{g,mf}) m_s$$



where

- $m_s = \rho_s \varepsilon_s$ mass per unit volume of the bed material particles
 k_a attrition rate constant, $k_a = (1.5 \pm 0.4) \times 10^{-4} \text{ m}^{-1}$
 H_e effective bed height
 d_h effective bed diameter
 $u_{g,mf}$ minimum superficial fluidization velocity of gas

and the *sphericity-dependent factor*

$$f(\phi_s) = \phi_s^{-2} \exp\left(\frac{(\phi_s^{0.75} - 1)^{1.75}}{\phi_s^2}\right)$$

Attrition 2

Combustion assisted attrition of fuel particles by bed material particles (Syamlal)

Particle **mass decrease rate**

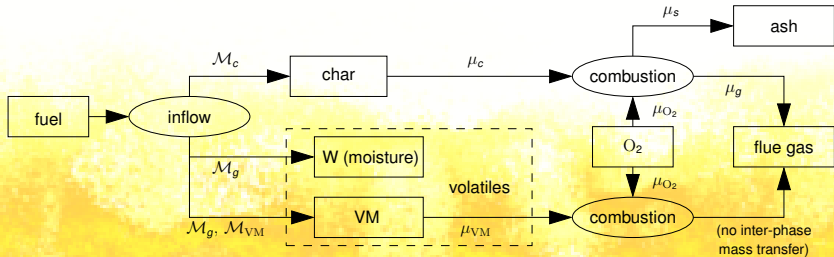
$$\frac{dm_i}{dt} = K'_a |u_s - u_i| A_i = \frac{K_{a,i}}{d_{p,i}} |u_s - u_i| m_i$$

for $i \in \{c, b\}$ where

- $K_{a,i} = \frac{6K'_a}{\rho_i}$ attrition rate constant (for spherical particles)
 K'_a .. modified attrition rate constant, $K'_a \approx 7.87 \times 10^{-5} \text{ kg} \cdot \text{m}^{-3}$
 A_i surface area of particles with total mass m_i

Fuel conversion

- **devolatilization:** volatiles treated as already released into the flue gas
- **volatiles burnout:** $\mu_{VM,c}$, $\mu_{VM,b}$... Arrhenian kinetic theory, depends on T , Y_{O_2}
- **char burnout:** μ_c , μ_b ... governed by oxygen diffusion to particle surface and by carbon oxidization rate (*Basu, Halder*)



Proximate and ultimate fuel analysis

| as received basis | | | | | | | | |
|-------------------|-------------------|--|----------------------|---|---|---|--------------|---|
| A | C | | | H | O | N | S | W |
| ash (A) | fixed carbon (FC) | | volatile matter (VM) | | | | moisture (W) | |
| char (coke) | | | volatiles | | | | | |

Oxygen consumption rate

$$\mu_{O_2} = \mu_c \cdot \nu_{\text{char}} + \mu_b \cdot \nu_{\text{char},b} + \mu_{\text{VM},c} \cdot \nu_{\text{VM},c} + \mu_{\text{VM},b} \cdot \nu_{\text{VM},b}$$

where ν_{char} , $\nu_{\text{VM},c}$, $\nu_{\text{VM},b}$ are the amounts of oxygen per unit mass of char, coal VM, and biomass VM

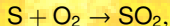
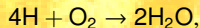
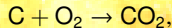
$$\nu_i = 32 \cdot \left(\frac{C_i}{12.01} + \frac{H_i}{4.032} + \frac{S_i}{32.06} \right) - O_i,$$

$$\nu_C = 32 \cdot \frac{1}{12.01},$$

$$\nu_{\text{char},i} = \frac{FC_i}{FC_i + A_i} \cdot \nu_C,$$

$$\nu_{\text{VM},i} = \frac{\nu_i - FC_i \cdot \nu_C}{\text{VM}}.$$

according to ultimate analysis C_i, H_i, O_i, N_i, S_i in as received basis and idealized combustion

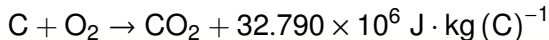


Heat production

- heating value of char from fuel $i \in \{c, b\}$

$$Q_i = \frac{FC_i}{FC_i + A_i} Q_{FC} = \frac{FC_i}{FC_i + A_i} \cdot 32.790 \times 10^6 \text{ J} \cdot \text{kg}^{-1}$$

according to the reaction



- LHV of VM released from fuel $i \in \{c, b\}$ is therefore

$$Q_{\text{LHV,VM},i} = \frac{Q_{\text{LHV},i} - FC_i \cdot Q_{FC}}{\text{VM}}$$

- **heat production** is given by the terms \dot{Q}_c and \dot{Q}_b which read

$$\dot{Q}_c(\mu_c, \mu_{\text{VM},c}) = Q_c \cdot \mu_c + Q_{\text{LHV,VM},c} \cdot \mu_{\text{VM},c},$$

$$\dot{Q}_b(\mu_b, \mu_{\text{VM},b}) = Q_b \cdot \mu_b + Q_{\text{LHV,VM},b} \cdot \mu_{\text{VM},b}.$$

Heat transfer

- **radiative** heat transfer:

$$\dot{Q}_R = \sigma (\epsilon(T) T^4 - \epsilon(T_{\text{wall}}) T_{\text{wall}}^4)$$

where ϵ is the flue gas emissivity

- **convective** heat transfer:

$$\dot{Q}_C = \frac{4}{D} \alpha (T - T_{\text{wall}})$$

where D is the combustor diameter and α the heat transfer coefficient

$$\alpha = \frac{\text{Nu} \cdot \lambda(T)}{L}$$

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Finite volume scheme

- aggregate vector form:

$$\frac{\partial \mathbf{W}}{\partial t} + \frac{\partial}{\partial x} (\mathbf{F}_c - \mathbf{F}_v) = \mathbf{Q}$$

\mathbf{W} vector of state variables
 $\mathbf{F}_c, \mathbf{F}_v$ convective and viscous fluxes

- semidiscrete scheme

$$\frac{d\mathbf{W}_K}{dt} = -\frac{1}{|K|} [(\mathbf{F}_{c,R} - \mathbf{F}_{v,R} + \mathbf{F}_{v,R}) - (\mathbf{F}_{c,L} - \mathbf{F}_{v,L} + \mathbf{F}_{v,L})] + \mathbf{Q}_K$$

R, L .. values at the left and right boundary of the cell K

- adjustable artificial diffusion \mathbf{F}_v used for stabilization
- explicit adaptive time integration (Runge-Kutta-Merson)
- parallel implementation written using C/C++ and the MPI library

Initial and boundary conditions

Boundary conditions

- **implementation:** auxiliary (ghost) cells beyond the computational domain
- **default:** zero Neumann b.c. (extrapolation) is imposed on all *physical* quantities
- **overriden at inlet:** Dirichlet b.c. for ε_i , $d_{p,i}$, T , and gas mass inflow $\rho_g \varepsilon_g U_g$
- **overriden at outlet:** Dirichlet b.c. for atmospheric pressure

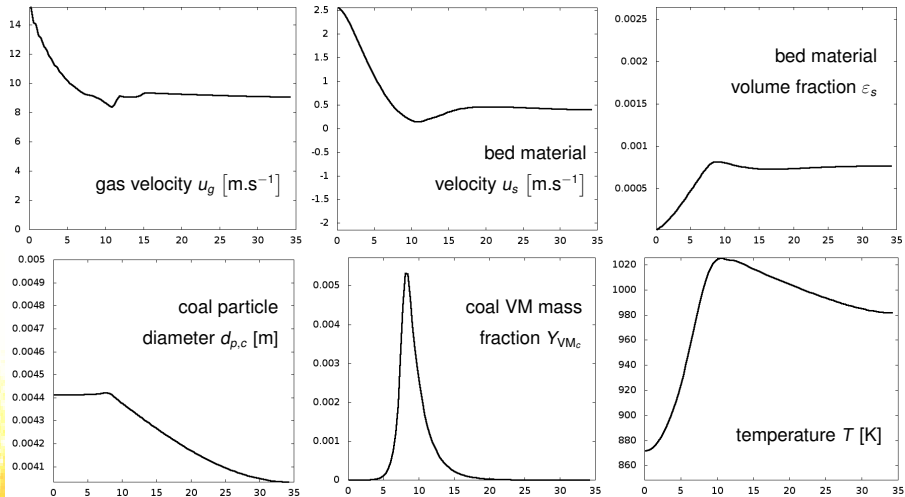
Initial conditions

- set up to quickly reach the stationary state
- zero flow velocities, atmospheric pressure, zero fuel inventory, and gas pre-heating

Solids recirculation

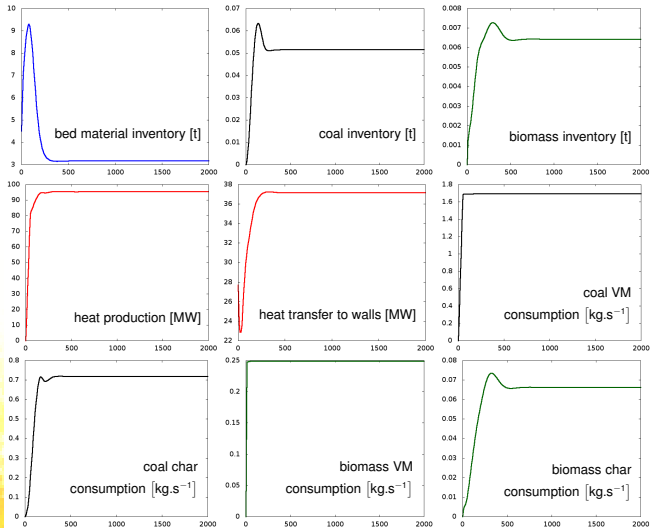
- a mixture of **bed material** (limestone and ash), **unburnt coal char**, and **unburnt biomass char**
- current **composition** of this mixture at the outlet used to calculate the composition of the injected material
- **rate of material injection** specified independently (large enough recirculation reservoir assumed)

Results



Vertical profiles of selected quantities at $t = 2000$ s

Results



Cumulative quantities $t \in \{0 \text{ s}, 2000 \text{ s}\}$